

Thermodynamic Analysis of Engineering Components

ME301 F2010 9.1

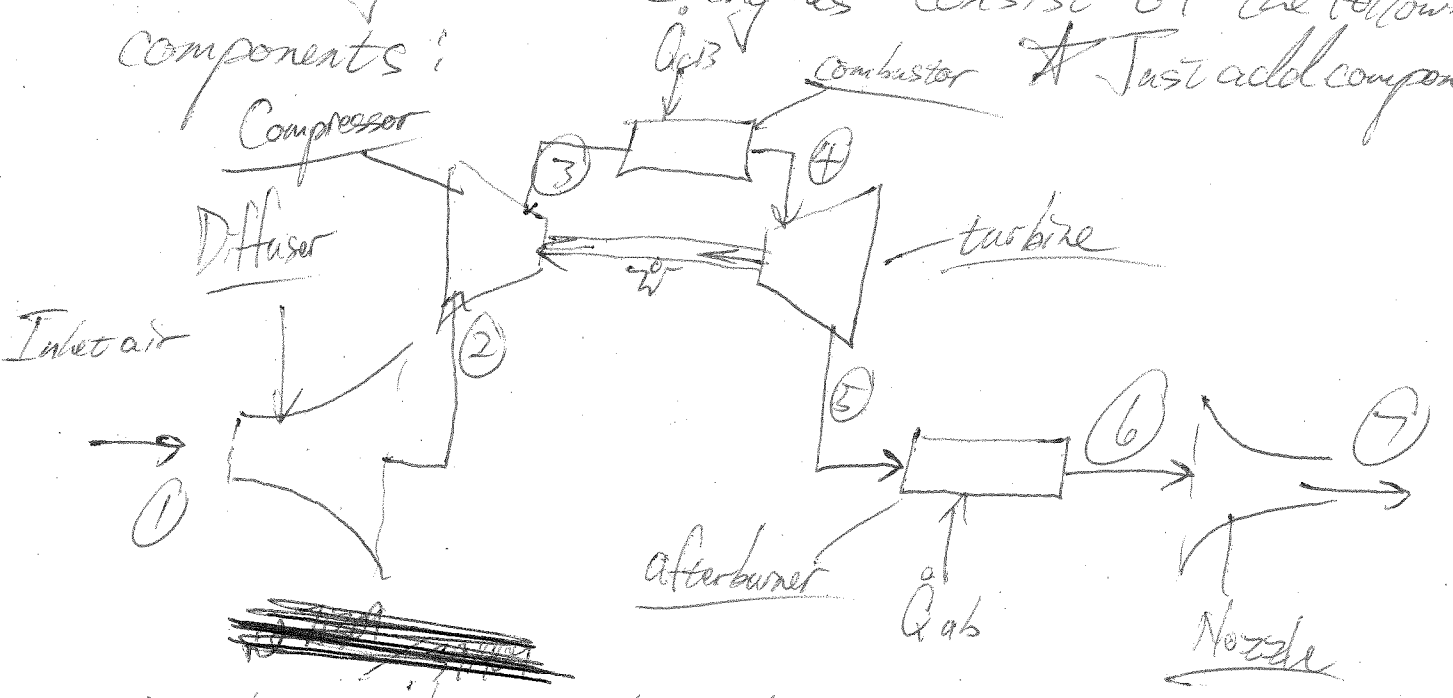
~~The plan for the syllabus was to cover pipe flow, nozzle flows, then move to comp~~

Now that we can apply energy balances to open systems, we can begin to look @ engineering devices:

We'll spend the next 2 days looking @ the Goblin II turbojet engine used on F80 military jet airplanes

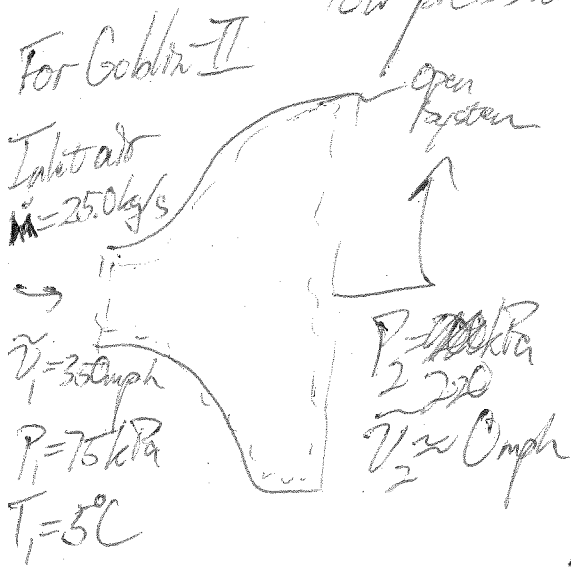
- Show picture

- Most ~~all~~ gas turbine engines consist of the following components:



We'll use what we've learned to analyze this system piece by piece...

Diffuser: The purpose of a diffuser is to convert a high velocity, low pressure flow into a low velocity, high pressure flow.



Assuming ideal with constant heat capacity, find a) Inlet diameter of Diffuser

b) ~~Temperature of air exiting diffuser~~

Mass balance (Rate): $\dot{M}_{in} = \dot{M}_{out} + \frac{dM}{dt}$ (steady state)

$\dot{M}_{in} = \dot{M}_{out} = 25.0 \text{ kg/s}$ how do we relate to A_{in} ?

$$\dot{M} \left(\frac{\text{kg}}{\text{s}} \right) = \rho A \bar{v} \quad \rho = \frac{1}{v_{m1}} \quad \text{where } v = \frac{RT}{P}$$

$$P_{air} = \frac{R_{ideal}}{m_{air}} = \frac{8314 \frac{\text{J}}{\text{kmol} \cdot \text{K}}}{28.97 \frac{\text{kg}}{\text{kmol}}} \Rightarrow P_{air} = 287.1 \frac{\text{J}}{\text{kg} \cdot \text{K}} \quad v_1 = \frac{287.1 \text{ J/kg} \cdot 278.15 \text{ K}}{75000 \text{ Pa}}$$

$$v_1 = 1.065 \frac{\text{m}^3}{\text{kg}}$$

$$\dot{M} = \frac{\rho D^2}{4} v_{in} (156.5 \text{ m/s}) = 25 \text{ kg/s} \Rightarrow D_{in} = 0.4654 \text{ m}$$

B) Temperature of air ~~etc~~ leaving the diffuser \rightarrow Assumptions: 1) Steady state, 2) Negligible, 3) No flow shaft work, 4) Adiabatic, 5) Ideal gas

Energy Balance: $\dot{E}_{in} = \dot{E}_{out} + \frac{dE}{dt} \rightarrow 0$ (steady state)

$$\dot{Q}_{in} + \dot{W}_{in} + \dot{M}_1 \left(h_1 + \frac{v_1^2}{2} + g z_1 \right) = \dot{Q}_{out} + \dot{W}_{out} + \dot{M}_2 \left(h_2 + \frac{v_2^2}{2} + g z_2 \right) \Rightarrow \dot{M}_1 \left(h_1 + \frac{v_1^2}{2} \right) = \dot{M}_2 \left(h_2 + \frac{v_2^2}{2} \right)$$

algebra $\dot{M} \frac{v_1^2}{2} = \dot{M} (h_2 - h_1)$ for ideal gas constant $C_p h = C_p (T_2 - T_1)$

$$\dot{M} \frac{v_1^2}{2} = \dot{M} C_p (T_2 - T_1) \Rightarrow \dot{M} \frac{v_1^2}{2} = \dot{M} C_p T_2 - \dot{M} C_p T_1$$

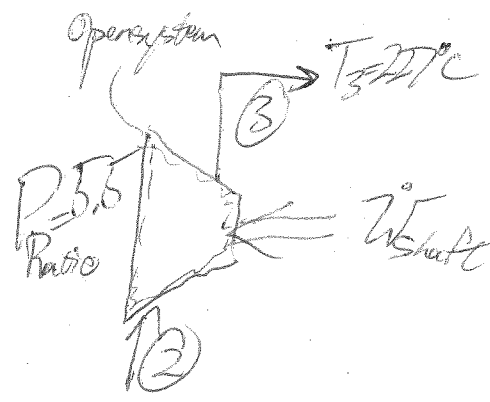
~~$T_2 = 250 \frac{1565 \frac{m}{s}}{1005 \frac{J}{kg \cdot K}} + 250 \frac{1005 \frac{J}{kg \cdot K}}{1005 \frac{J}{kg \cdot K}} (278.15 K)$~~ $\Rightarrow T_2 = 290.3 K$

Flow heated? Makes sense!

$$T_2 = \frac{1}{C_p} \frac{\tilde{v}_1^2}{2} + T_1$$

1565
1005 J/kg

Compressor: The purpose of a compressor is to use work to increase the pressure of a fluid.



c) Determine the volumetric flow rate ~~of~~ the compressor.

Mass Balance: $\dot{M}_{in} = \dot{M}_{out}, \dot{M}_2 = \dot{M}_3 = 25 \text{ kg/s}$

$\dot{V}_3 = \frac{\dot{m}_3}{\rho} = \frac{\text{kg}}{\text{s}} \frac{\text{m}^3}{\text{kg}} = \text{m}^3/\text{s}$

$T_2 = 290.3 K$
 $P_2 = 220 \text{ kPa}$
 $\dot{M} = 25 \text{ kg/s}$
 $\tilde{v}_2 = 0$

$$V_3 = \frac{RT_3}{P_3} \quad \frac{P_3}{P_2} = P_{ratio} \Rightarrow P_3 = P_2 P_{ratio} \Rightarrow V_3 = \frac{RT_3}{P_2 P_{ratio}}$$

$$V_3 = \frac{287.1 \frac{J}{kg \cdot K} (500.2 K)}{5.5 (220,000 \text{ Pa})} \Rightarrow V_3 = \frac{287.1 \cdot 500.2}{5.5 \cdot 220,000} \text{ m}^3/\text{s}$$

$C_p = 1187 \text{ J/kg}$

$$\dot{V}_3 = 25 \text{ kg/s} \left(\frac{1}{1187 \text{ kg/m}^3} \right) \Rightarrow \dot{V}_3 = 2.967 \text{ m}^3/\text{s}$$

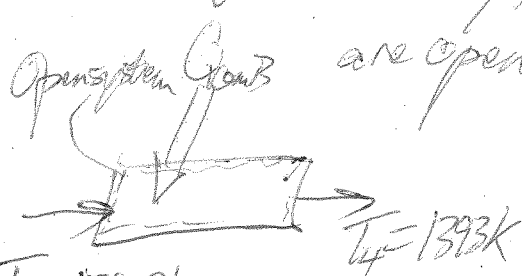
D) Find the power required by the compressor \rightarrow Energy Balance

$$\dot{E}_{in} = \dot{E}_{out} + \dot{W}_{shaft} \Rightarrow \dot{Q}_{in} + \dot{W}_{in} + \dot{M}_2 \left(h_2 + \frac{\tilde{v}_2^2}{2} + g z_2 \right) = \dot{Q}_{out} + \dot{W}_{out} + \dot{M}_3 \left(h_3 + \frac{\tilde{v}_3^2}{2} + g z_3 \right)$$

$\dot{W}_{in} = \dot{M} (h_3 - h_2)$ \leftarrow general equation for a compressor Assumptions: 1) SS, 2) KE, PE negligible, 3) adiabatic

$$\dot{W}_{in} = \dot{M} c_p (T_3 - T_2) \Rightarrow 25 \frac{\text{kg}}{\text{s}} \cdot 1005 \frac{\text{J}}{\text{kg} \cdot \text{K}} (500.2 \text{ K} - 290.3 \text{ K}) \Rightarrow \dot{W}_{in} = 5,272,000 \text{ W}$$

Combustor: The purpose of a combustor is to add heat to a flow of fluid. Sometimes just the heat is specified. Sometime rate of fuel & heating value of fuel is specified. Usually, they are large combustion chambers that are open at either end. This means they are isobaric $P_3 = P_4$.



c.) Find the heat transfer that must be added to the combustor.

$T_3 = 500.2K$
 $\dot{M}_3 = 25 \text{ kg/s}$
 $P_3 = 1,210,000 \text{ Pa}$

Mass Balance: $\dot{M}_{in} = \dot{M}_{out}$ $\dot{M}_3 = \dot{M}_4$

- Assumptions:
- 1) Steady State
 - 2) $\Delta PE, \Delta KE$ negligible
 - 3) No shaft or PV work
 - 4) ideal constant C_p

Energy Balance: $\dot{E}_{in} = \dot{E}_{out} + \frac{dE}{dt}$

$$\dot{Q}_{in} + \dot{W}_{in} + \dot{M}_3 \left(h_3 + \frac{v_3^2}{2} + gz_3 \right) = \dot{Q}_{out} + \dot{W}_{out} + \dot{M}_4 \left(h_4 + \frac{v_4^2}{2} + gz_4 \right)$$

$\dot{Q}_{in} = \dot{M}_4 (h_4 - h_3)$ ← general equation for combustor with above assumptions for ideal gas (constant C_p)

$$\Rightarrow \dot{Q}_{in} = \dot{M} C_p (T_4 - T_3) \Rightarrow 25 \frac{\text{kg}}{\text{s}} (1005 \frac{\text{J}}{\text{kg} \cdot \text{K}}) (1393K - 500.2K)$$

$25 \frac{\text{kg}}{\text{s}} (1575.42 - 503.02) = 25,310,000 \text{ W}$

$\dot{Q}_{comb} = 22,449,000 \text{ W}$

Noticing a pattern?

Problem solving steps: Included

- 1) ~~Make a drawing & note the known property/state information.~~
- 2) ~~Choose the system to solve for important specifications with feasible assumptions.~~
- 3) ~~Apply mass balance with assumptions to simplify.~~
- 4) ~~Apply energy balance with assumptions to simplify.~~
- 5) ~~Apply fluid model: real (EES or tables), ideal (constant or not C)~~
- 6) ~~Solve resulting set of equations~~